

Introduction to evaporative pre-cooling

June 6, 2023

Safety minute and house keeping items

Food and beverages

- Help yourself throughout
- Garbage cans in the back

Restrooms

Through the atrium

Emergency protocols

- Emergency exits
- Meeting location/find a Platte River employee

Upcoming events

- July Efficiency Works Business tour more information available soon
- August Fort Collins exterior lighting code updates
- Quarterly Selling Energy webinars for listed service providers with licenses Let us know if you'd like to become a listed service provider and receive a license
- Service provider 1-on-1's Send us an email to schedule
- November Service provider social and awards

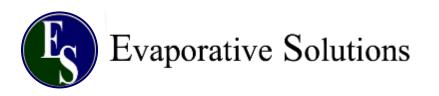
Register for Efficiency Works events: https://efficiencyworks.org/resources/events/



Bryan Curtis

Evaporative Solutions





Providing the Highest Quality PreCooling Systems for Air-Cooled Condensers





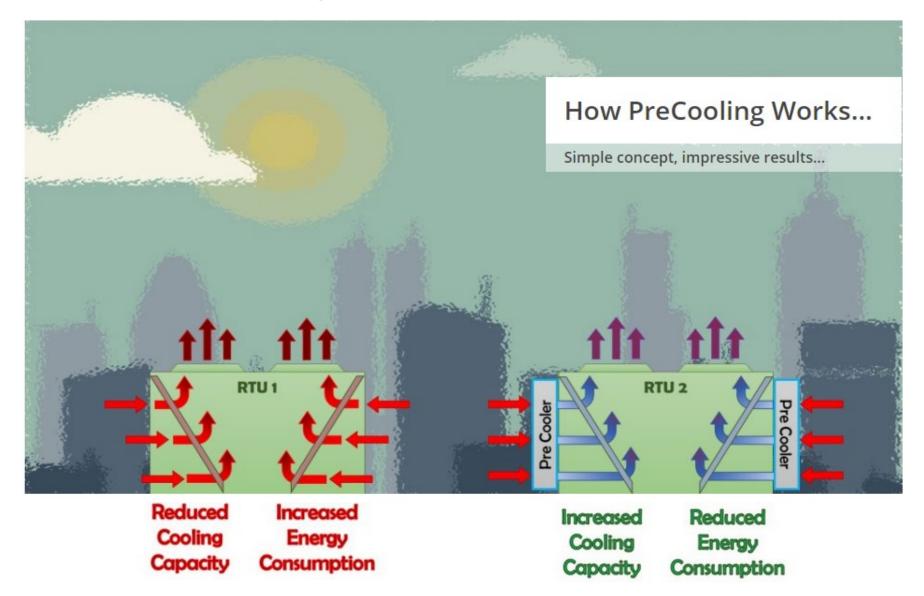








Providing the Highest Quality PreCooling Systems for Air-Cooled Condensers







Impact of Elevated Ambient Temperatures on Capacity and Energy Input to a Vapor Compression System – Literature Review

Letter report for ARTI 21-CR Research Project: 605-50010/605-50015

S. Yana Motta and Piotr A. Domanski National Institute of Standards and Technology Gaithersburg, MD





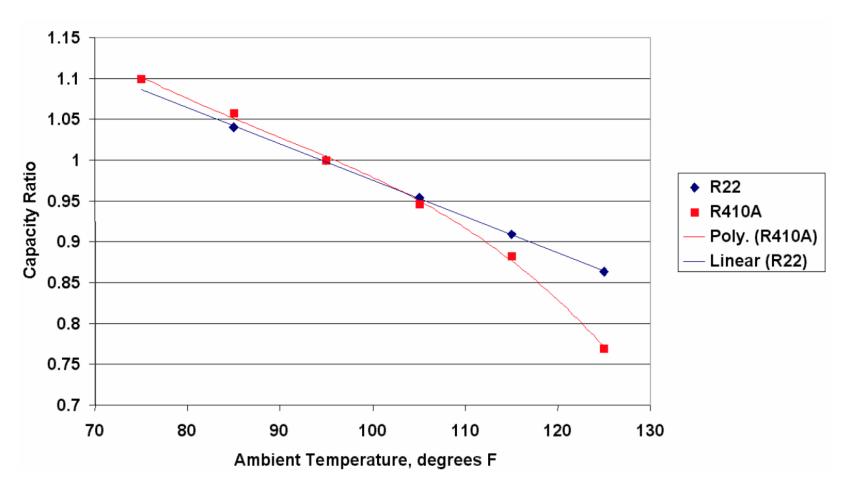


Figure 5. Comparison of capacity loss versus ambient temperature, split system A/C, 12-13 SEER (Wells et al., 1999).





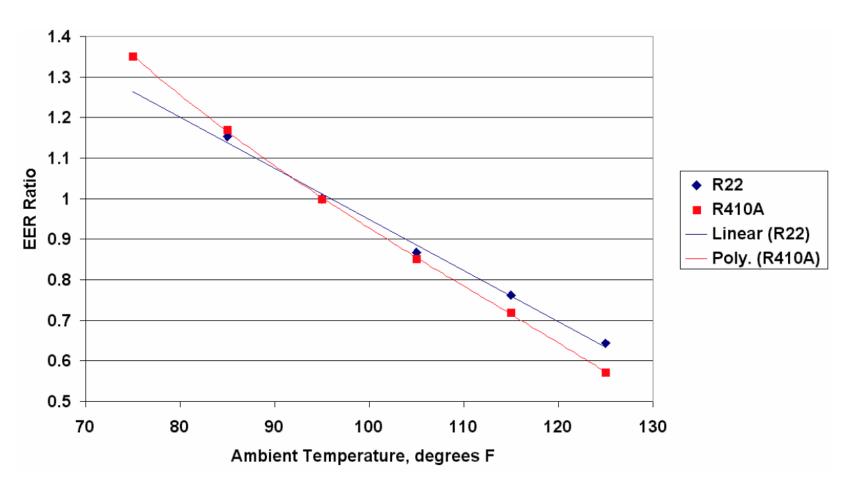


Figure 6. Comparison of EER loss versus ambient temperature, split system A/C, 12-13 SEER (Wells et al., 1999).





An EDISON INTERNATIONAL® Company

Performance Evaluation of Rooftop Air Conditioning Units At High Ambient Temperatures

Ramin Faramarzi, Bruce Coburn, Rafik Sarhadian, Scott Mitchell, and R. Anthony Pierce, Southern California Edison





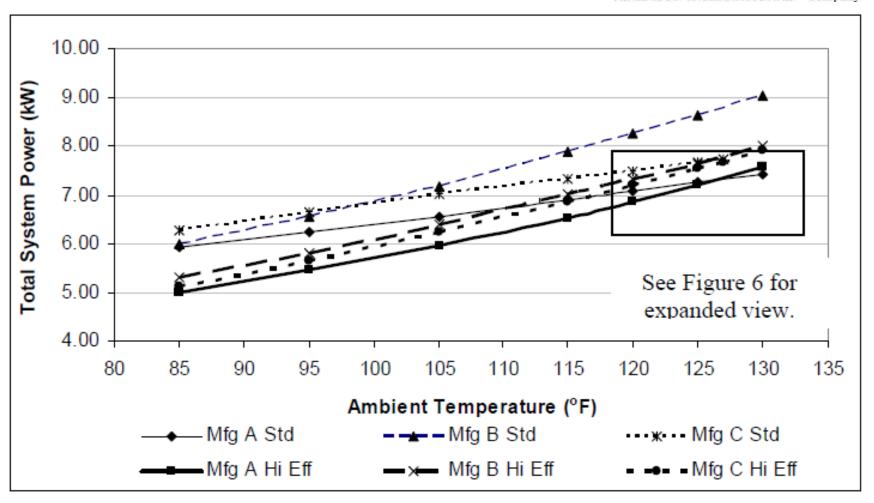


Figure 5. Total System Power Consumption Based on RTTC Test Data for All Six Standard and High Efficiency Units Subject to Various Ambient Temperatures





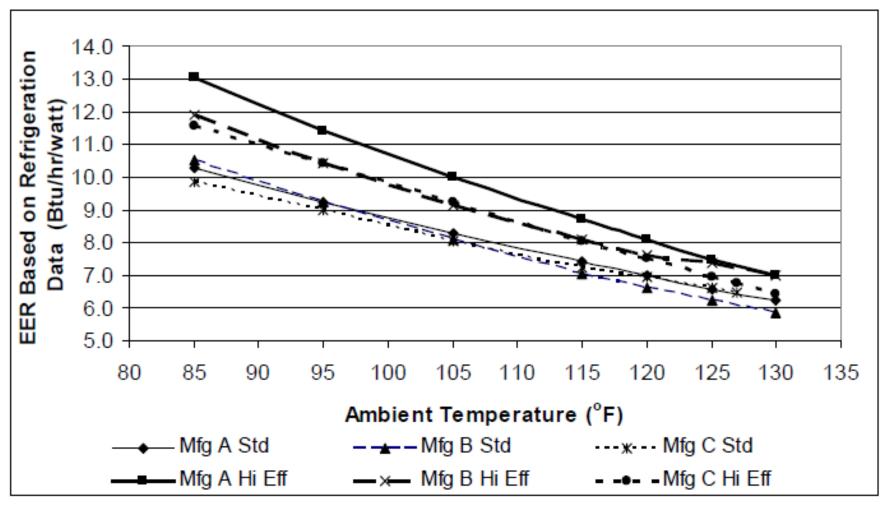
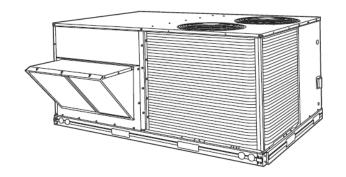


Figure 8. EERs Based on RTTC Refrigeration Side Test Data for All Six Standard and High Efficiency Units Subject to Various Ambient Temperatures





Small Packaged Unit Performance Data (12.5 Tons)



| 50HJ014 (121/2 TONS) | | | | | | | | | | | | |
|----------------------|-----------|--|---------------|----------------|----------------|----------------|---------------|--|--|--|--|--|
| LCWT (F) | | Air Temperature Entering Condenser (F) | | | | | | | | | | |
| | | 75 | 85 | 95 | 105 | 115 | 125 | | | | | |
| | TC | 142.0 | 135.6 | 126.3 | 115.7 | 104.4 | 93.9 | | | | | |
| 45 | SHC kW | 123.6 8.95 | 120.9 9.97 | 116.5 11.01 | 111.2 12.09 | 104.2 13.22 | 93.8 14.44 | | | | | |

Increase in Tons

11%

Decrease in KW

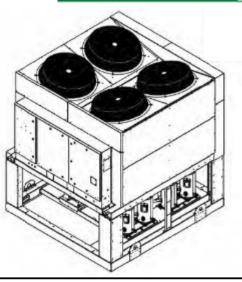
19%

Increase in SHC





Mid-size Scroll Compressor Chiller Performance Data (50 Ton)



| | Ambient Air Temperature | | | | | | | | | | | | | | |
|------|-------------------------|-------|------|------|-------|------------|--------------|-------|------|---------------|-------|------|-------|-------|------|
| LWT | LWT 75°F | | | 85°F | | | 95° F | | | 105° F | | | 115°F | | |
| (°F) | Unit | Power | Unit | Unit | Power | Unit | Unit | Power | Unit | Unit | Power | Unit | Unit | Power | Unit |
| | Tons | kWi | ₽ | Tons | kWi | ⊞ R | Tons | kWi | ₩ | Tons | kWi | Ħ | Tons | kWi | ₽ |
| 44 | 53.8 | 46.7 | 13.8 | 51.1 | 51.5 | 11.9 | 48.1 | 57.1 | 10.1 | 45.0 | 63.4 | 8.5 | 41.6 | 70.6 | 7.1 |

Increase in Tons

11%

Decrease in KW

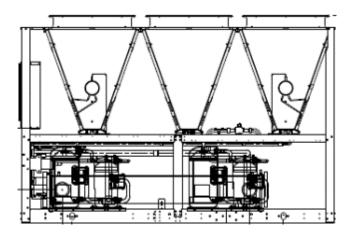
18%

Increase in EER





Large Scroll Compressor Chiller Performance Data (100 Ton)



| MODEL: YLAA0100SE | | | | | | | | | | | | IPLV= 14.3 | | | | | | |
|-------------------|-------------------------------------|------|------|-------|------|------|-------|------|------|------|-------|------------|------|-------|-----|------|-------|-----|
| | AIR TEMPERATURE ON - CONDENSER (°F) | | | | | | | | | | | | | | | | | |
| LCWT | | 75.0 | | | 80.0 | | | 85.0 | | | 90.0 | | | 95.0 | | | 100.0 | |
| (°F) | TONS | KW | EER | TONS | KW | EER | TONS | KW | EER | TONS | KW | EER | TONS | KW | EER | TONS | KW | EER |
| 44.0 | 107.6 | 89.3 | 13.2 | 104.7 | 94.2 | 12.3 | 101.8 | 99.3 | 11.3 | 98.9 | 104.8 | 10.5 | 95.8 | 110.7 | 9.6 | 92.1 | 116.5 | 8.8 |

Increase in Tons

11%

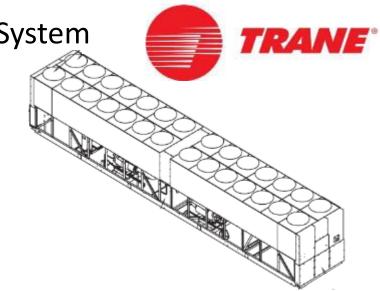
Decrease in KW

19%

Increase in EER



Air-Cooled Rotary Liquid Chiller Performance Data (500 Ton)



| Table P-1. 60 Hz | able P-1. 60 Hz standard efficiency machines in English units | | | | | | | | | | | | |
|--|---|-------|--|------|-------|----------|-----|-------|----------|-----|-------|----------|-----|
| | | | Condenser Entering Air Temperature (F) | | | | | | | | | | |
| | | 85 | | | | 95 | | 105 | | | 115 | | |
| Evaporator Leaving Water Temperature (F) | Unit Size Model RTAC | Tons | kW input | EER | Tons | kW input | EER | Ton | kW input | EER | Tons | kW input | EER |
| 44 | 500 STD | 515.8 | 519.3 | 11.0 | 483.0 | 560.6 | 9.6 | 448.8 | 607.4 | 8.3 | 413.3 | 659.7 | 7.1 |

Increase in Tons

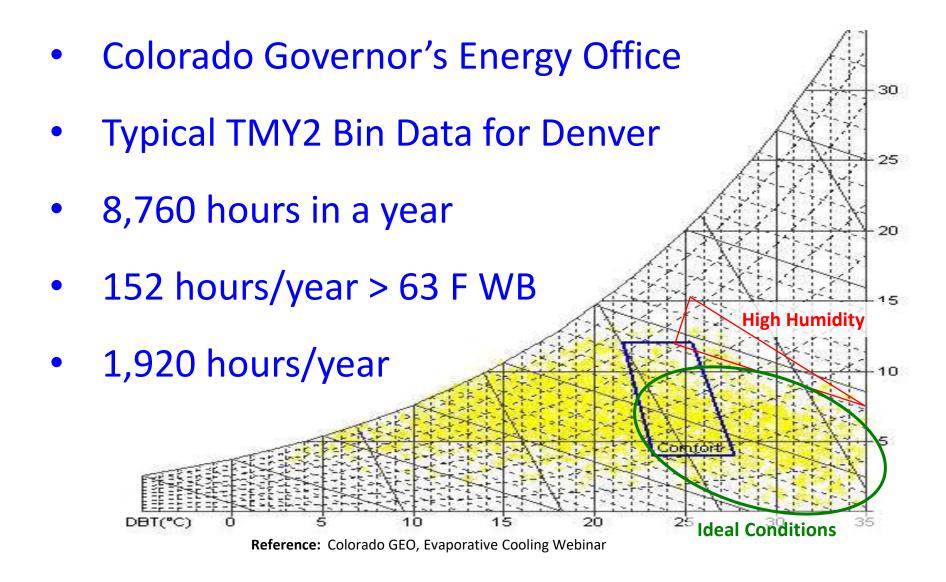
13%

Decrease in KW

15%

Increase in EER

Denver's Climate on the Psychrometric Chart





Many chiller manufacturers offer adiabatic precooling as a factory installed option











What NOT to do...





What NOT to do continued...





...and this is why.

There is a better way!









But what about the water usage!?!?!?!?!?

We are in a drought!!!!

How much water will be wasted!?!?!

What about the cost of the water!?!?!?



Evaporative PreCooling Actually Saves Water!



...and Money!

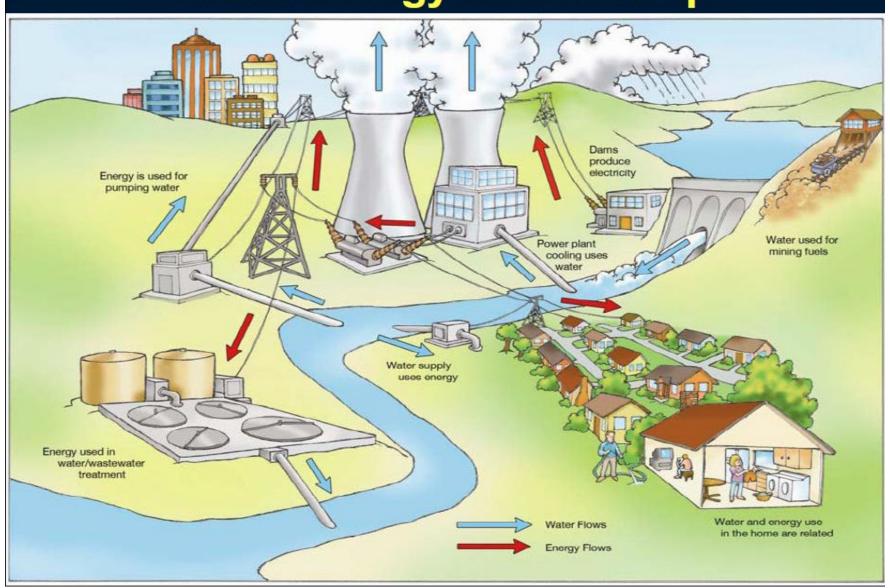


Critical Natural Resources

- Water and Energy are both Critical Resources
- Water and Energy Production are Interrelated
 - Water is needed to produce Energy
 - Energy is needed to store and transport Water



Water Energy Relationship





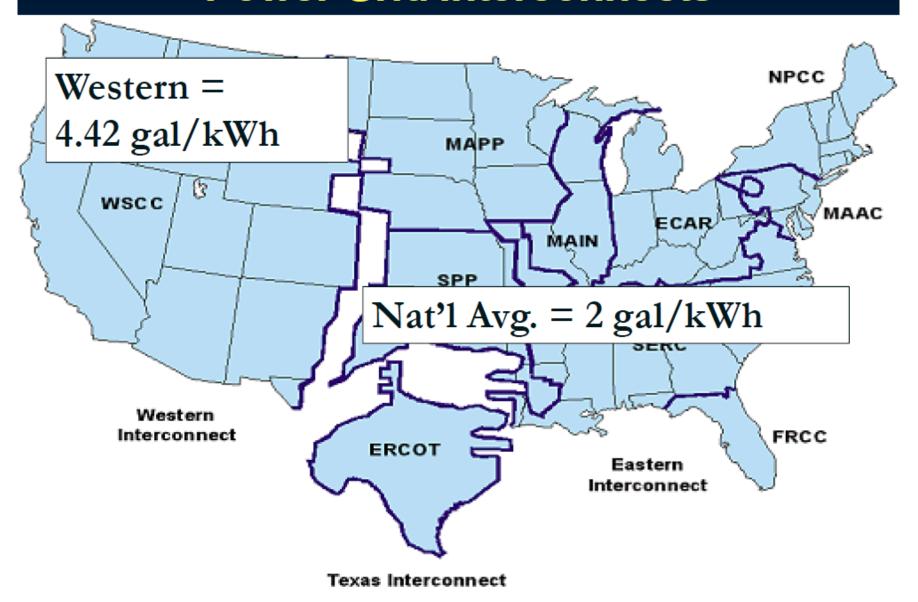
Resource Relationship: Water in Energy

 2003 National Renewable Energy Lab report:

On average 2 gallons of water consumed per 1 kWh electricity generated in the US

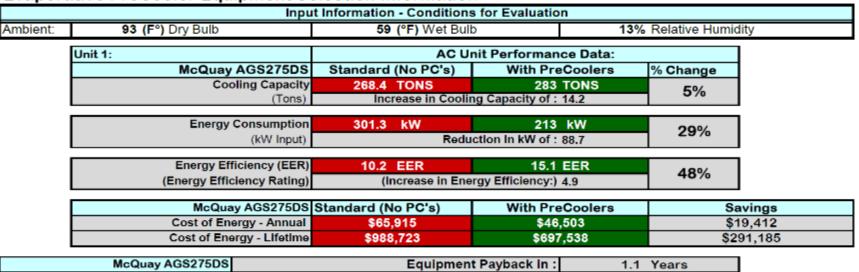


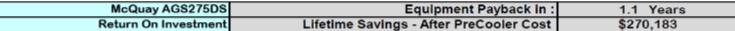
Power Grid Interconnects

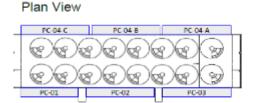




Evaporative PreCooler Equipment Selection Information









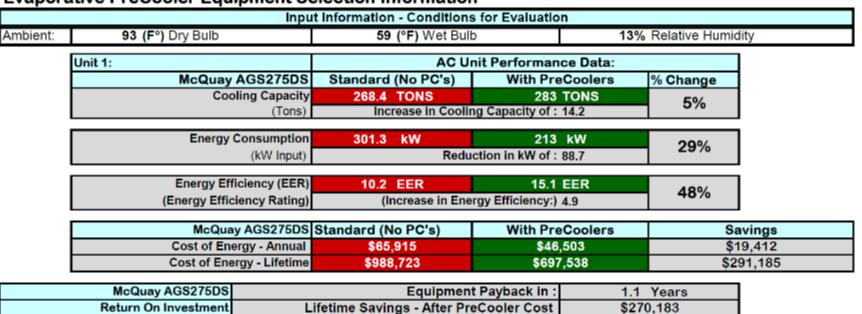


Water Saved by PreCoolers 4.42 Gallon per kWh 176473 kWh Saved Water Saved 780009 344,636 Water Used Net Water Savings 435,373



Evaporative PreCooler Equipment Selection Information

PC-03



| Plan View | Elevation View 1 | Elevation View 2 | | | | | |
|-------------------------|---------------------------|---|--|--|--|--|--|
| PC-04-C PC-04-B PC-04-A | PC-01 PC-02 PC-03 94 x 84 | PC-04-A PC-04-B PC-04-C 96 x 78 96 x 78 | | | | | |

| Additional PreCooler Operating Costs: kWh Saved = 176,473 | | | | | | | | |
|---|---------|------------------|-----------------|--|--|--|--|--|
| Water Consumption | 344,636 | Gallons per Year | | | | | | |
| Cost/1000 Gallons | \$4.85 | Denver Water | Commercial Rate | | | | | |
| Water Cost | \$1,672 | Annual | | | | | | |

@ 2023 Rates =

\$23,471

Less Water Costs =

\$21,799

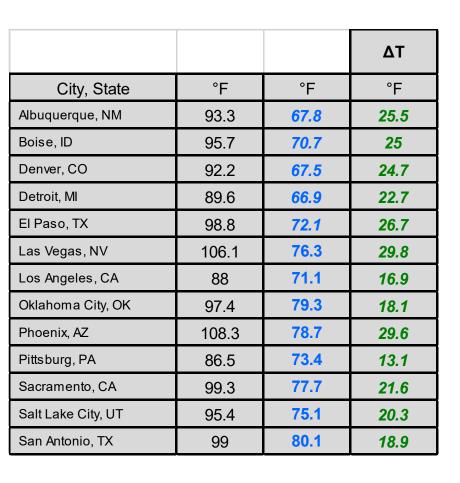


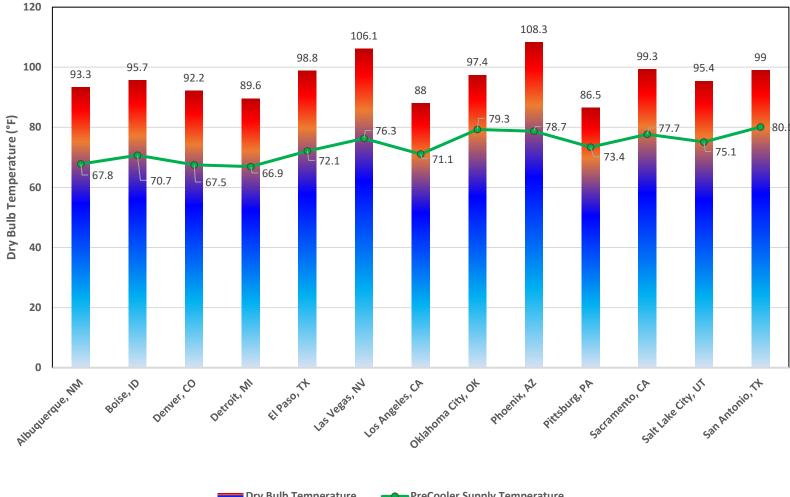


Evaporative PreCooling Performance Data

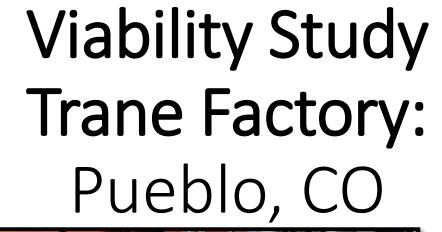
Dry Bulb Temperature Differential with Evap PreCoolers

(2017 ASHRAE 1% Design Dry Bulb vs PreCooler Supply Temperature)









AHRI Approved
Test Facility



Test Unit: Trane CGAM 140-Std

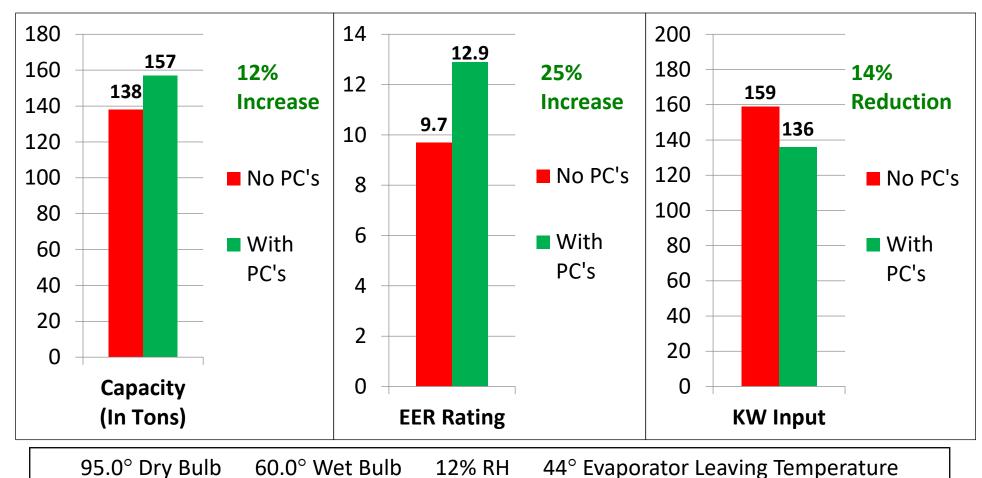
Hydr**EVAP** Specs:

- ✓ 7" Housing
- √ 4" Media
- ✓ No Block-Off Panels





Test Results – 12%RH



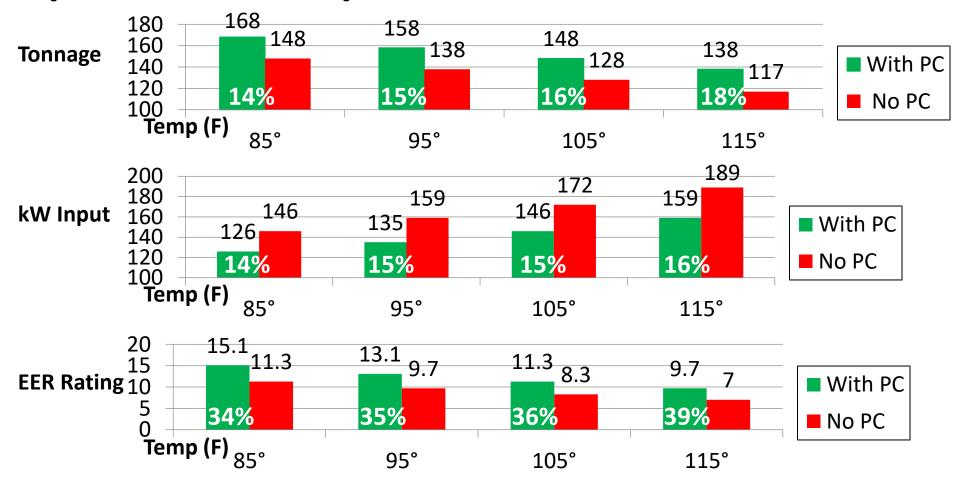
12% RH

44° Evaporator Leaving Temperature

60.0° Wet Bulb



Temperature Specific Data – 12%RH





Evaporative PreCooling Performance Data



Trane Factory: Pueblo, CO

"We were able to confirm benefit of the product...and we concluded that this is a viable product for the arid portions of the US, and we would like to promote it at that region."

-Todd Duncan Product Manager Scroll & Screw Chillers Trane





Performance Data Provided by Client

Date: 6/22 - 6/29/2012

Location: Highlands Ranch,

Colorado

Client: Douglas County

Ave. Temp: 93° F

Ave Humidity: 23% RH

Unit: Trane Intellipak

(40 Ton Unit)

On average that is nearly a 40% savings in kWh!

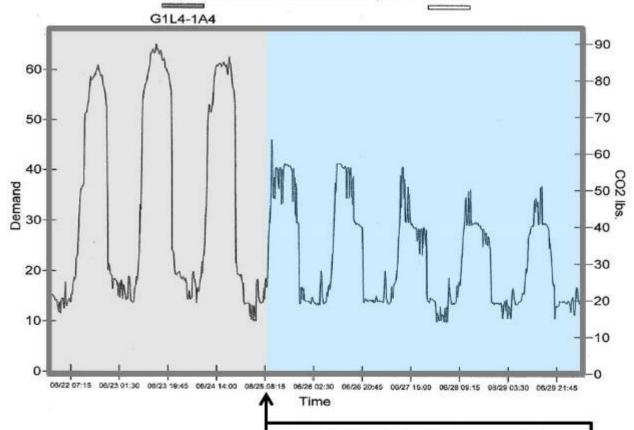












Hydr**EVAP** PreCoolers Installed



Performance Data - HydrEVAP



Chiller Service Report

Without PreCoolers













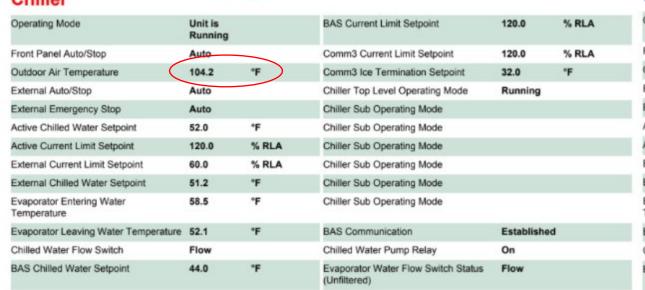




Wednesday, July 15, 2020 2:23:47 PM PDT

Chiller Status

Chiller



Wednesday, July 15, 2020 2:40:30 PM PDT

Chiller Status

Chiller



Chiller Service Report With PreCoolers





Performance Data - HydrEVAP Evaporative PreCoolers



| Circuit 1 | Without | PreCoo | lers | | | Circuit 1 | With Pr | eCooler | "S | | |
|---|-------------------|---------------|---|---------|---------------|---|-------------------|---------------|---|---------|---------------|
| Circuit 1 Sub Mode | | | Air Flow | 85.7 | % | Circuit 1 Sub Mode | | | Air Flow | 85.7 | % |
| Circuit 1 Sub Mode | | | Inverter Speed | 0.0 | % FullSpeed | Circuit 1 Sub Mode | | | Inverter Speed | 0.0 | % FullSpeed |
| Circuit 1 Sub Mode | | | Condenser Refrigerant Pressure | 266.8 | psi gauge | Circuit 1 Sub Mode | | | Condenser Refrigerant Pressure | 170.8 | psi gauge |
| Circuit 1 Sub Mode | | | Saturated Condenser Refrigerant Temperature | 151.0 | °F calculated | Circuit 1 Sub Mode | | | Saturated Condenser Refrigerant Temperature | 119.8 | *F calculated |
| Circuit 1 Sub Mode | | | Differential Refrigerant Pressure | 222.7 | psid | Circuit 1 Sub Mode | | | Differential Refrigerant Pressure | 128.0 | psid |
| Circuit 1 Sub Mode | | | Evaporator Refrigerant Pressure | 43.7 | psi gauge | Circuit 1 Sub Mode | | | Evaporator Refrigerant Pressure | 42.8 | psi gauge |
| Circuit 1 Top Level Operating Mode | Running | | Saturated Evaporator Refrigerant Temperature | 48.3 | °F calculated | Circuit 1 Top Level Operating Mode | Running | | Saturated Evaporator Refrigerant Temperature | 47.4 | °F calculated |
| External Hardwired Lockout | Not Locked Out | l | EXV Position | 38.5 | % Open | External Hardwired Lockout | Not Locked Out | | EXV Position | 44.2 | % Open |
| Front Panel Lockout | Not Locked Out | | Evaporator Refrigerant Liquid Level | -0.1 | in | Front Panel Lockout | Not Locked Out | | Evaporator Refrigerant Liquid Level | -0.2 | in |
| Circuit 2 | | | | | | Circuit 2 | | | | | |
| External Hardwired Lockout | Not Locked Out | | EXV Position | 100.0 | % Open | External Hardwired Lockout | Not Locked Out | | EXV Position | 83.0 | % Open |
| Front Panel Lockout | Not Locked Out | | Evaporator Refrigerant Liquid Level | -0.3 | in | Front Panel Lockout | Not Locked Out | | Evaporator Refrigerant Liquid Level | -0.0 | in |
| Air Flow | 100.0 | % | Circuit 2 Top Level Operating Mode | Running | | Air Flow | 100.0 | % | Circuit 2 Top Level Operating Mode | Running | |
| Inverter Speed | 100.0 | % FullSpeed | Circuit 2 Sub Mode | | | Inverter Speed | 100.0 | % FullSpeed | Circuit 2 Sub Mode | | |
| Condenser Refrigerant Pressure | 177.1 | psi gauge | Circuit 2 Sub Mode | | | Condenser Refrigerant Pressure | 114.2 | psi gauge | Circuit 2 Sub Mode | | |
| Saturated Condenser Refrigerant Temperature | 122.2 | °F calculated | Circuit 2 Sub Mode | | | Saturated Condenser Refrigerant Temperature | 95.1 | F calculated | Circuit 2 Sub Mode | | |
| Differential Refrigerant Pressure | 132.5 | psid | Circuit 2 Sub Mode | | | Differential Refrigerant Pressure | 68.7 | psid | Circuit 2 Sub Mode | | |
| Evaporator Refrigerant Pressure | 44.8 | psi gauge | Circuit 2 Sub Mode | | | Evaporator Refrigerant Pressure | 45.1 | psi gauge | Circuit 2 Sub Mode | | |
| Saturated Evaporator Refrigerant Temperature | 49.3 | °F calculated | Circuit 2 Sub Mode | | | Saturated Evaporator Refrigerant Temperature | 49.6 | °F calculated | Circuit 2 Sub Mode | | |







Compressor 1A Without PreCoolers

| Run Hours | 11187:29 | hrs:mins | Intermediate Oil Pressure | 248.6 | psi gauge |
|------------------------|----------|----------|---|---------|-----------|
| Starts | 1,929 | | Female Step Loader | Load | |
| Phase A-B Voltage | 462 | volts | High Pressure Cutout Switch | Good | |
| Average Line Current | 97.1 | % RLA | Compressor 1A Operating Mode | Running | |
| Line 1 Current | 193.0 | amps | Compressor 1A Top Level Operating Mode | Running | |
| Line 2 Current | 200.0 | amps | Compressor 1A Sub Mode | | |
| Line 3 Current | 188.0 | amps | Compressor 1A Sub Mode | | |
| Line 1 Current | 96.6 | % RLA | Compressor 1A Sub Mode | | |
| Line 2 Current | 100.0 | % RLA | Compressor 1A Sub Mode | | |
| Line 3 Current | 94.3 | % RLA | Compressor 1A Sub Mode | | |
| Maximum Line Current | 100.0 | % RLA | Compressor 1A Sub Mode | | |
| Supply Oil Temperature | 189.6 | °F | | | |

Compressor 1B

| Compressor 12 | | | | | |
|------------------------|----------|----------|---|---------|-----------|
| Run Hours | 11252:47 | hrs:mins | Intermediate Oil Pressure | 248.0 | psi gauge |
| Starts | 1,923 | | Female Step Loader | Load | |
| Average Line Current | 82.5 | % RLA | High Pressure Cutout Switch | Good | |
| Line 1 Current | 163.0 | amps | Compressor 1B Operating Mode | Running | |
| Line 2 Current | 171.0 | amps | Compressor 1B Top Level Operating Mode | Running | |
| Line 3 Current | 161.0 | amps | Compressor 1B Sub Mode | | |
| Line 1 Current | 81.6 | % RLA | Compressor 1B Sub Mode | | |
| Line 2 Current | 85.5 | % RLA | Compressor 1B Sub Mode | | |
| Line 3 Current | 80.6 | % RLA | Compressor 1B Sub Mode | | |
| Maximum Line Current | 85.1 | % RLA | Compressor 1B Sub Mode | | |
| Supply Oil Temperature | 189.5 | °F | Compressor 1B Sub Mode | | |
| | | | | | |

Compressor 1A With PreCoolers

| Run Hours | 11187:40 | hrs:mins | Intermediate Oil Pressure | 159.8 | psi gauge |
|------------------------|----------|----------|---|---------|-----------|
| Starts | 1,929 | | Female Step Loader | Load | |
| Phase A-B Voltage | 465 | volts | High Pressure Cutout Switch | Good | |
| Average Line Current | 60.7 | % RLA | Compressor 1A Operating Mode | Running | |
| Line 1 Current | 120.0 | amps | Compressor 1A Top Level Operating Mode | Running | |
| Line 2 Current | 125.0 | amps | Compressor 1A Sub Mode | | |
| Line 3 Current | 118.0 | amps | Compressor 1A Sub Mode | | |
| Line 1 Current | 59.9 | % RLA | Compressor 1A Sub Mode | | |
| Line 2 Current | 63.0 | % RLA | Compressor 1A Sub Mode | | |
| Line 3 Current | 59.1 | % RLA | Compressor 1A Sub Mode | | |
| Maximum Line Current | 63.1 | % RLA | Compressor 1A Sub Mode | | |
| Supply Oil Temperature | 146.1 | °F | | | |
| | | | | | |

Compressor 1B

| Run Hours | 11252:58 | hrs:mins | Intermediate Oil Pressure | 160.0 | psi gauge |
|------------------------|----------|----------|---|---------|-----------|
| Starts | 1,923 | | Female Step Loader | Load | |
| Average Line Current | 61.0 | % RLA | High Pressure Cutout Switch | Good | |
| Line 1 Current | 120.0 | amps | Compressor 1B Operating Mode | Running | |
| Line 2 Current | 126.0 | amps | Compressor 1B Top Level Operating Mode | Running | |
| Line 3 Current | 119.0 | amps | Compressor 1B Sub Mode | | |
| Line 1 Current | 60.1 | % RLA | Compressor 1B Sub Mode | | |
| Line 2 Current | 63.3 | % RLA | Compressor 1B Sub Mode | | |
| Line 3 Current | 59.8 | % RLA | Compressor 1B Sub Mode | | |
| Maximum Line Current | 63.3 | % RLA | Compressor 1B Sub Mode | | |
| Supply Oil Temperature | 146.1 | °F | Compressor 1B Sub Mode | | |







Compressor 2A Without PreCoolers

| Run Hours | 11316:07 | hrs:mins | Intermediate Oil Pressure | 166.3 | psi gauge |
|------------------------|----------|----------|---|---------|-----------|
| Starts | 1,916 | | Female Step Loader | Unload | |
| Average Line Current | 55.2 | % RLA | High Pressure Cutout Switch | Good | |
| Line 1 Current | 107.0 | amps | Compressor 2A Operating Mode | Running | |
| Line 2 Current | 115.0 | amps | Compressor 2A Top Level Operating Mode | Running | |
| Line 3 Current | 107.0 | amps | Compressor 2A Sub Mode | | |
| Line 1 Current | 53.5 | % RLA | Compressor 2A Sub Mode | | |
| Line 2 Current | 57.7 | % RLA | Compressor 2A Sub Mode | | |
| Line 3 Current | 53.5 | % RLA | Compressor 2A Sub Mode | | |
| Maximum Line Current | 58.1 | % RLA | Compressor 2A Sub Mode | | |
| Supply Oil Temperature | 159.8 | °F | Compressor 2A Sub Mode | | |
| | \sim | | | | |

Compressor 2A With PreCoolers

| - | | | | | |
|------------------------|----------|----------|---|---------|-----------|
| Run Hours | 11316:17 | hrs:mins | Intermediate Oil Pressure | 46.1 | psi gauge |
| Starts | 1,916 | | Female Step Loader | Unload | |
| Average Line Current | 0.0 | % RLA | High Pressure Cutout Switch | Good | |
| Line 1 Current | 0.0 | amps | Compressor 2A Operating Mode | Stopped | |
| Line 2 Current | 0.0 | amps | Compressor 2A Top Level Operating Mode | Auto | |
| Line 3 Current | 0.0 | amps | Compressor 2A Sub Mode | | |
| Line 1 Current | 0.0 | % RLA | Compressor 2A Sub Mode | | |
| Line 2 Current | 0.0 | % RLA | Compressor 2A Sub Mode | | |
| Line 3 Current | 0.0 | % RLA | Compressor 2A Sub Mode | | |
| Maximum Line Current | 0.0 | % RLA | Compressor 2A Sub Mode | | |
| Supply Oil Temperature | 122.9 | °F | Compressor 2A Sub Mode | | |
| | | | | | |

Compressor 2B

| Run Hours | 11206:49 | hrs:mins | Intermediate Oil Pressure | 164.6 | psi gauge | R |
|------------------------|----------|----------|---|---------|-----------|----|
| Starts | 1,926 | | Female Step Loader | Load | | S |
| Average Line Current | 75.1 | % RLA | High Pressure Cutout Switch | Good | | A |
| Line 1 Current | 148.0 | amps | Compressor 2B Operating Mode | Running | | Li |
| Line 2 Current | 155.0 | amps | Compressor 2B Top Level Operating Mode | Running | | Li |
| Line 3 Current | 145.0 | amps | Compressor 2B Sub Mode | | | Li |
| Line 1 Current | 74.4 | % RLA | Compressor 2B Sub Mode | | | Li |
| Line 2 Current | 78.0 | % RLA | Compressor 2B Sub Mode | | | Li |
| Line 3 Current | 72.9 | % RLA | Compressor 2B Sub Mode | | | Li |
| Maximum Line Current | 78.1 | % RLA | Compressor 2B Sub Mode | | | M |
| Supply Oil Temperature | 159.8 | °F | Compressor 2B Sub Mode | | | S |
| | | | | | | |

Compressor 2B

| Run Hours | 11207:00 | hrs:mins | Intermediate Oil Pressure | 106.5 | psi gauge |
|------------------------|----------|----------|---|---------|-----------|
| Starts | 1,926 | | Female Step Loader | Load | |
| Average Line Current | 55.6 | % RLA | High Pressure Cutout Switch | Good | |
| Line 1 Current | 109.0 | amps | Compressor 2B Operating Mode | Running | |
| Line 2 Current | 115.0 | amps | Compressor 2B Top Level Operating Mode | Running | |
| Line 3 Current | 108.0 | amps | Compressor 2B Sub Mode | | |
| Line 1 Current | 54.7 | % RLA | Compressor 2B Sub Mode | | |
| Line 2 Current | 58.0 | % RLA | Compressor 2B Sub Mode | | |
| Line 3 Current | 54.1 | % RLA | Compressor 2B Sub Mode | | |
| Maximum Line Current | 57.9 | % RLA | Compressor 2B Sub Mode | | |
| Supply Oil Temperature | 126.4 | °F | Compressor 2B Sub Mode | | |



Performance Data - HydrEVAP Evaporative PreCoolers











Wednesday, July 15, 2020 2:23:47 PM PDT

Chiller Status

Chiller



Chiller Service Report
Without PreCoolers

Wednesday, July 15, 2020 2:40:30 PM PDT

Chiller Status
Chiller



Chiller Service Report
With PreCoolers

Tons: 453.5 Tons: 537.0 Increase of 83.5 Tons + 18%

kW: 1,492.4 kW: 853.5 Reduction of 638.9 kW - 43%

kW/Ton: 3.29 kW/Ton: 1.58 Reduction of 1.7 kW/Ton - 52%



Performance Data - HydrEll







Wednesday, July 15, 2020 2:40:30 PM PDT

Chiller Status Chiller

Chiller Service Report With PreCoolers

x 6 units



Like having a 7th Chiller! Increase of 501 Tons!

Reduction of 3,833.4 kW! **During peak demand!**

Reach setpoint without running 6 compressors!



ATEC/Indirex

Tel: (303) 816-7075 Fax: (800) 859-5592

Info@haveacoolday.com

Sales Representative: Erik Jeanette

Quotation: HEQ103017 Arvada West High School Chiller

Hydr**EVAP**

Contact: Maggie Anderson

Project: Avada West High School Chiller

Date: 30-Oct-17

Model: McQuay AGS275DS

Location: Arvada, CO

Evaporative PreCooler Equipment Selection Information

| | Inpu | ıt Information - Conditions for Evaluatio | n |
|----------|------------------|---|-----------------------|
| Ambient: | 93 (F°) Dry Bulb | 59 (°F) Wet Bulb | 13% Relative Humidity |

| Unit 1: | AC Unit Performance Data: | | | | | | |
|------------------|---|--|----|--|--|--|--|
| McQuay AGS275DS | Standard (No PC's) With PreCoolers % Change | | | | | | |
| Cooling Capacity | 268.4 TONS | 283 TONS | 5% | | | | |
| (Tons) | Increase in Coolii | Increase in Cooling Capacity of : 14.2 | | | | | |

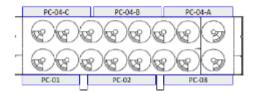
| Energy Consumption | 301.3 | kW | 213 kW | 29% |
|--------------------|-------|-----|-------------------------|------|
| (kW Input) | | Red | luction in kW of : 88.7 | 29/0 |

| Energy Efficiency (EER) | 10.2 EER | 15.1 EER | 48% |
|----------------------------|------------------|----------|-----|
| (Energy Efficiency Rating) | (Increase in Ene | 40% | |

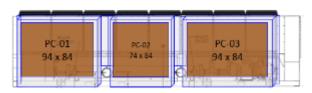
| McQuay AGS275DS | Standard (No PC's) | With PreCoolers | Savings |
|---------------------------|--------------------|-----------------|-----------|
| Cost of Energy - Annual | \$65,915 | \$46,503 | \$19,412 |
| Cost of Energy - Lifetime | \$988,723 | \$697,538 | \$291,185 |

| McQuay AGS275DS | Equipment Payback in : | 1.1 Years |
|----------------------|---|-----------|
| Return On Investment | Lifetime Savings - After PreCooler Cost | \$270,183 |

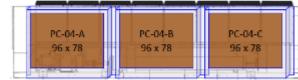
Plan View



Elevation View 1



Elevation View 2



| | | | 1 | I . | | | | |
|------------------|---------------------------------|-------------|-----------------------------------|---------------|---------------|----|-------------|-----|
| | PreCooler P | rice | Optional Equ | ipment | | Ch | iller | |
| ID# | Per Chiller | | Per Chiller | | Qty | Su | bTotal | Ш |
| Unit 1 | \$ 2 | 27,744.43 | \$ | 3,757.83 | 1 | \$ | 31,502.26 | |
| McQuay AGS275DS | Total Est. Shipping Weight: 535 | | | | | - | | |
| _ | Notes: | | | | | | | |
| | \$ 1 | 16,995.00 | | | 1 | \$ | 16,995.00 | |
| Installation | Notes: Insta | llation doe | s not include | providing wat | er source or |] | | |
| | electrical red | quirements | to chiller vici | nity. | | | | |
| | \$ | 1,395.00 | | | 5 | \$ | 6,975.00 | |
| Annual Service | Notes: Sprin | g start-up, | , Mid-Summe | r check, Fall | shut-down. |] | | |
| | Replacemen | t pumps ir | ncluded, exclu | udes media r | eplacement. | | | _ |
| Efficiency Works | \$ (4 | 11,250.00) | | | 1 | \$ | (41,250.00) | 1 |
| rebate with 50% | Standard + | Bonus reb | ate \$150/nom | ninal ton | | | | 7 |
| bonus | | | | | | | | |
| | | | | | | | | |
| | | | | | | | | |
| | | | | | | | | Ш |
| | | | | | | | | |
| | | | | | | | | 6 |
| | | | | | | | | |
| | | | | PreCoo | ler SubTotal | \$ | 14,222.26 | |
| | | | | | | | | |
| # of Pallets | 2 | | | | zing Charge | _ | 250.00 | (8) |
| Total Weight | 785 | | Est. Sh | ipping/Handl | ing Charges | \$ | 1,000.00 | |
| | | | | | | | | 28 |
| | | | | | | | | 100 |
| | | | Quote FOB: | | co | \$ | 15,472.26 | |
| | PreCoolers, | Shipping, | and Optional | Equipment | | | | |
| | | | | | | | | |
| | | | | | ıal Savings: | | 19,412.00 | |
| | | | Estimated ROI (In Years/Seasons): | | | _ | 8.0 | |
| | | | Estimated ROI (In Months): | | | | 3.5 | |
| | | | Estimated Li | fetime Savi | ngs (15 yrs): | \$ | 275,707.74 | F |
| | | | | | | | | T |
| | | | | | | | | T |







Efficiency Works rebates

Evaporative Condensing



Evaporative pre-cooling incentives

| Equipment upgrade | Description Incentive | | Limited time incentive | | |
|---------------------------|---|---------|------------------------|-------|---------|
| Evaporative Condensing | Evaporative media or mist to pre-cool air entering the condenser of a rooftop unit (RTU) or air-cooled chiller. In the unlikely event that the evaporative equipment damages a condenser or part of a condenser, and it is less than 15 years old, the manufacturer shall replace the condenser or damaged part of the condenser and pay for the cost of the study to determine the cause of failure. | \$100 p | per ton | \$150 | per ton |

- For a limited time, a 50% bonus is available for cooling rebates. Projects must be completed and submitted for payment by November 15, 2023.
- Preapproval is required for all projects with incentives greater than \$10,000
- Incentives are limited to total project cost for projects with incentives less than \$50,000





Questions?

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Thank you for participating in Efficiency Works Business

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<u>EfficiencyWorks.org</u>